

ENERGY SAVING IN BUILDING WITH LATENT HEAT STORAGE

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Abstract

This article presents an experimental analysis of peak load shifting for air conditioning system using PCM (phase change material) in a room. The melting range of the used PCM is from 20 to 25°C. The amount of the PCM is determined according to the cooling load of the room during the peak time. The hourly cooling load is calculated. A low-power fan is used to drive air over the PCM capsules, which are arranged in a chamber above the room. Auxiliary cooling unit is used to freeze the PCM when the night temperature is higher than 18°C. The temperatures of the PCM are measured by 9 thermocouples inserted inside a capsule. The variant degrees of temperature of the air at the inlet, at the middle, and at the outlet of the air-passing chamber as well as at the room are measured during day and night. In addition to that, the climate conditions are measured. As a result, during the peak load shifting time, which is within 2 hours, the decrease of the room temperature is between 7-10 °C by using PCM ceiling system. Consequently, it can be concluded that the PCM system is effective for the peak load shifting.

Keywords: *peak load shifting - PCM thermal storage- air-conditioning*

1. Introduction

As the demand for air conditioning increased greatly during the last decade, there has been a large demand of electric power. In addition, the limited reserves of fossil fuels have led to a surge of interest in efficient energy application. Electrical energy consumption varies significantly during the day and night according to the demand by industrial, commercial and residential activities. In hot climate countries, the major part of the load variation is due to air conditioning. Better power management and significant

economic benefit can be achieved if some of the peak load could be shifted to the off-peak load period. This target can be achieved by thermal energy storage for cooling in residential and commercial building establishments. Phase change materials (PCMs) have been considered for thermal storage in buildings since 1980 [1]. There are several promising developments are taking place in the field of application of PCMs for heating and cooling of building. Experimental study of two real size concrete cubicles concluded that the energy storage in the walls by encapsulating PCMs and the

comparison with conventional concrete without PCMs led to an improved thermal inertia as well as lower inner temperatures [1]. Pasupathy and Velraj studied the thermal performance of an inorganic eutectic PCM based thermal storage system for thermal management in a residential building by theoretical and experimental investigation [2]. Performance of a hybrid heating-system, combined with PCM thermal storage was investigated numerically [3]. The results indicated that the thermal storage of PCM plates improved the indoor thermal comfort level and saved about 47% of normaland- peak-hour energy and 12% of total energy consumption in winter in Beijing. The wallboards are cheap and widely used in a variety of applications, making them very suitable for PCM encapsulation. To improve the wallboard efficiency, a vacuum insulation panel (VIP) was associated to the PCM panel [4]. An empirical model for a real-scale prototype of a PCM-air heat exchanger was built from experimental results, aimed at simulating the thermal behavior in the tested heat exchanger in different cases [5]. The use of the granular PCM can lead to improvement of the indoor thermal environment in comparison with that in conventional systems due to thermal radiation from the floor surface area, which can be maintained around the phase change temperature [6]. Studies of the free-cooling potential for different climatic locations were investigated [7]. It was found that the optimum PCM had a melting temperature that was approximately equal to the average

ambient air temperature in the hottest month, and that the free-cooling potential was proportional to the average daily amplitude of the ambient air's temperature swings. For all the analyzed climatic conditions, the PCM with a wider phase change temperature range (12 K) was found to be the most efficient. In experiments, a PCM wallboard room was constructed by attaching PCM wallboards, developed by incorporating about 26% PCM by weight into gypsum wallboards, to the surface of an ordinary wall [8]. Compared with an ordinary room, it was found that the PCM wallboard room could greatly reduce the energy cost of HVAC systems and transfer electric power peak load to valley. Experimental analysis of cooling buildings using nighttime cold accumulation in paraffin with a melting point of 22 °C as the PCM was investigated [9].

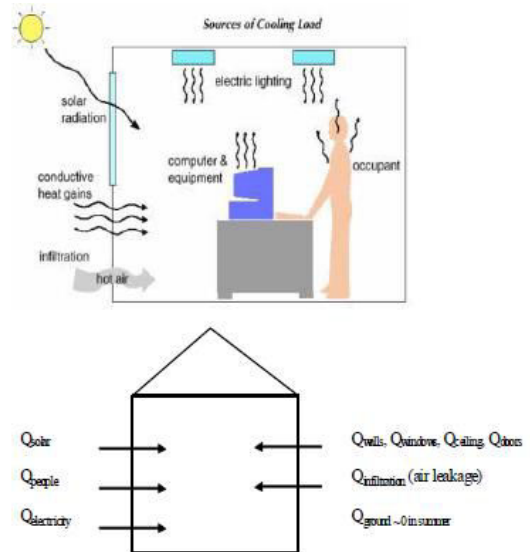


Fig. 1 The major sources of heat gain into a building [16]

2. Experimental work

2.1. Thermal design for PCM mass calculations

The PCM storage capacity is designed to be used in the test room with its thermal loads during the peak time. To reach appropriate thermal comfort the room temperature should be in the range between 21 and 28 °C. Consequently, to overcome the day/night time lag problem the thermal storage capacity of the PCM has to accommodate the heat gains within the space during the peak time. The net heat load into a building is called the cooling load. The total building cooling load consists of heat transferred through the building envelope (walls, roof, floor, windows, doors etc.) and heat generated by occupants, equipment, and lights. The major sources of heat gain into a building are shown in Fig. 1 [16]. Note that ground losses/gains are typically small and are here assumed negligible. The load due to heat transfer through the envelope is called as external load, while all other loads are called as internal loads. Based on energy balance, the net cooling load is calculated as follows:

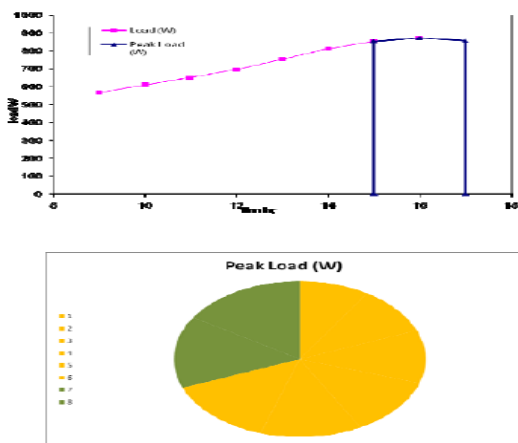
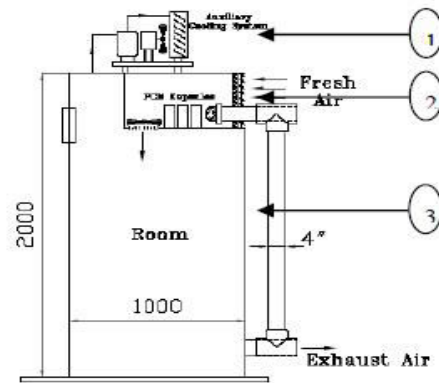


Fig. 2 Calculated peak load

2.2. Outline of the system

The result from the thermal design process is used to produce prototype module for latent heat thermal storage for the laboratory test room. As shown in Fig. 3 the experimental apparatus consists of the following:

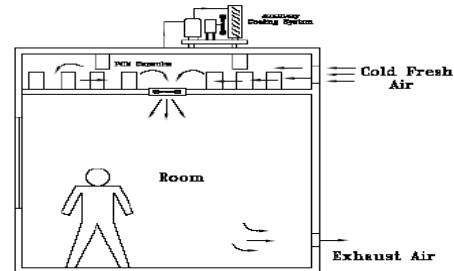
- i) Auxiliary refrigeration circuit.
- ii) PCM chamber of capsules.
- iii) Room space.



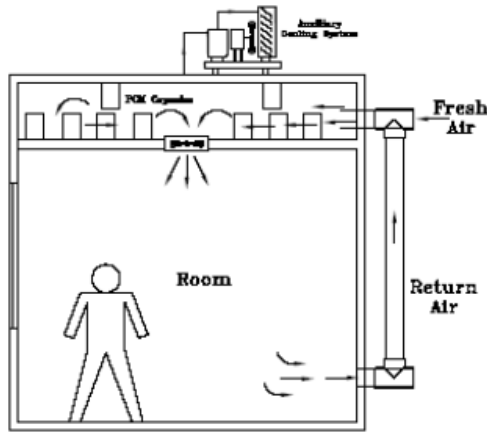
a- Schematic of the prototype



b- Photo of the prototype



a- PCM capsules cooled at night by fresh air (charge)



b. Room cooled by PCM (discharge)

2.3. Measurements

The data of solar radiation on horizontal surface and ambient air temperature are measured by using weather station located in the field of experiments in Cairo. The Data Logger is DL2e of DELTA-T DEVICES (Solar radiation sensor measuring range from -0.100000 to 1.499429kW/m²) (Ambient air temperature sensor measuring range -19.150000 to 60.000000°C). The PCM-capsule's dimensions are 125 mm in height and 100 mm in diameter.

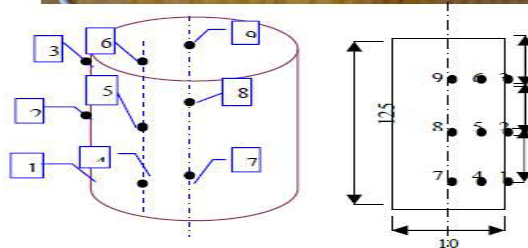


Fig. 5 The PCM Capsule and thermocouple's positions

3. Results and Discussion

3.1. Weather condition

Figure 6 shows the data of the average ambient air temperature for each month at the field of the experiments in 2010. The mean maximum and minimum temperatures are shown in table 2. From the table, it appears that for more than 120 days, PCM Melting at 22-24oC is workable for free cooling. 6 months are marked in green background and 6 months in white background. From May to October free-cooling can not be made available and electric cooling can be used for load shifting regime. PCM melting at 22-24oC will easily give comfortable temperature during day time for rest of the months.

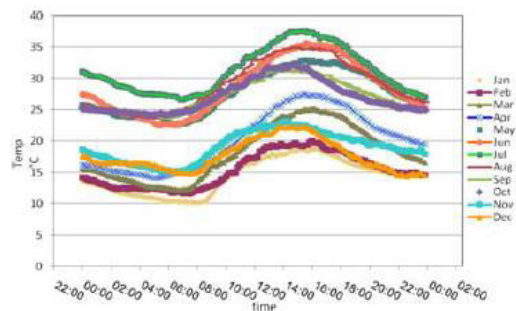


Fig. 6 Average ambient air temperature for each month at the field of the experiments in 2010

Table 2 The mean maximum and minimum temperatures

Month	Mean Temperature °C		Remarks
	Daily Min	Daily Max	
Jan	10.2	19.3	26-28°C can be maintained
Feb	11.7	19.9	26-28°C can be maintained
Mar	12.2	25.2	26-28°C can be maintained
Apr	14.1	27.4	26-28°C can be maintained
May	22.8	32.9	PCM Cooling can be used
Jun	22.6	35.6	PCM Cooling can be used
Jul	26.6	37.5	PCM Cooling can be used
Aug	23.6	34.9	PCM Cooling can be used
Sep	23.6	31.9	PCM Cooling can be used
Oct	24.1	32.3	PCM Cooling can be used
Nov	15.0	23.2	26-28°C can be maintained
Dec	14.8	22.3	26-28°C can be maintained

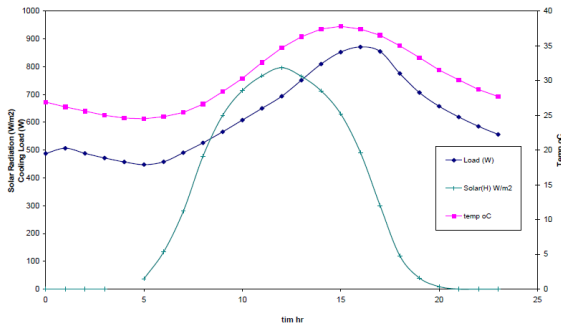


Fig. 7 Hourly cooling load, solar radiation, and the ambient air temperature

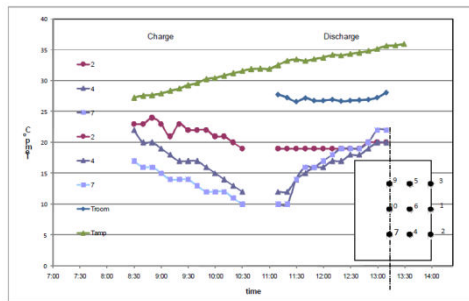


Fig. 8 Temperature profiles of the PCM (at the bottom)

process, the conduction through the bottom wall of the capsule causes rapid cooling at the bottom.

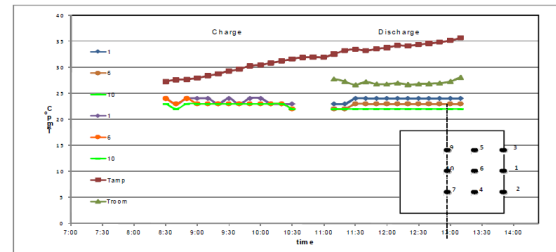


Fig. 9 Temperature profiles of the PCM (at the middle)

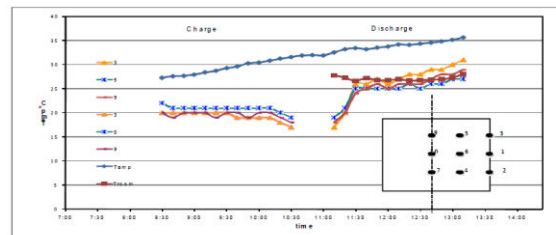


Fig. 10 Temperature profiles of the PCM (at the top)

3.3 Temperature profiles

The radial temperature profiles of the PCM at three different heights (30 mm, 60 mm, and 90 mm) during the charge and discharge periods are shown in Figs. 8, 9, and 10. The same figures show the variation of the ambient air temperature and the room temperature. The positions of the thermocouples are at the center of the PCM capsule and at radii of 25mm and 50mm. The temperature curves illustrate the temperature distribution in three regions during the charge and the discharge processes. The liquid region is above the upper limit of the melting range. The phase transition region is within the melting range. The solid region is below the lower limit of the melting range. At bottom of the capsule, as shown in Fig. 8, visible variation in temperature is observed during the charge and discharge processes. During the charge

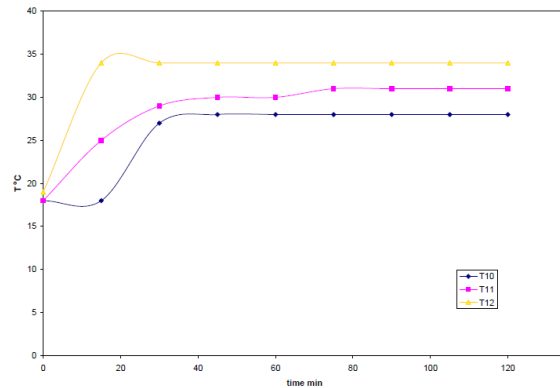


Fig. 11 Air temperature inside the duct

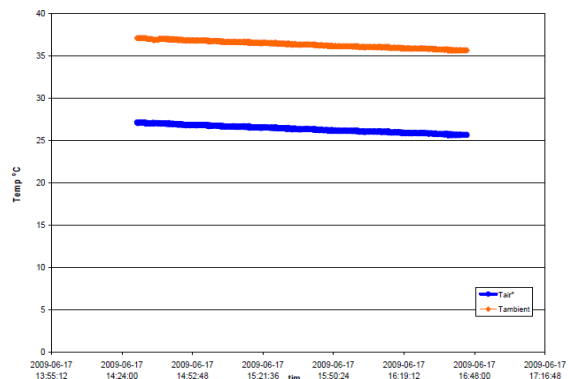


Fig. 12 Experimental results of the room air-temperature

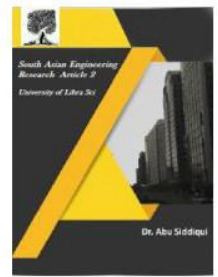


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5. Conclusions

In this research, the effects of the peak shaving control using PCM capsules in a ceiling-chamber for a room are examined, which is located in Cairo. The following results were obtained by examining the designed model experimentally.

1. In summer the cooling load peak is at 4 p.m. due to the time that heat takes to be conducted through the walls.
2. PCM melting at 22-24°C is workable for free cooling for 6 months. For the rest 6 months the peak shaving control can be applied by a night electric cooling unit.
3. Within 2 hours of peak load shifting time, the room can be kept cooled at a comfort temperature by cooling with off-peak energy.
4. Using free cooling of PCM can save 30% of the energy used for air conditioning.
5. Using free cooling of PCM can maintain the temperature within the room between 20 to 26°C.
6. The change in the air temperature in storage chamber is proportional to the change in the PCM temperature inside the capsules.
7. The change in the air temperature in the room is proportional to the change of ambient air temperature.
8. The COP of the system is about 2.3. From these results, it can be concluded that the PCM ceiling system is effective for the peak shaving control. Large amounts of latent heat can be stored in the PCM, which enables to make more compatible, clean, durable, and cheaper systems for air conditioning.

References

- [1] Luisa F. Cabeza, Cecilia Castello'n , Miquel Nogue's, Marc Medrano, Ron Leppers, Oihana Zubillaga, Use of microencapsulated PCM in concrete walls for energy savings, *Energy and Buildings* 39 (2007) 113–119.
<http://dx.doi.org/10.1016/j.enbuild.2006.03.030>
- [2] Pasupathy, R. Velraj, Effect of double layer phase change material in building roof for year round thermal management, *Energy and Buildings* 40 (2008) 193-203.
<http://dx.doi.org/10.1016/j.enbuild.2007.02.016>
- [3] Guobing Zhou, Yiping Zhang, Qunli Zhang, Kunping Lin, Hongfa Di, Performance of a hybrid heating system with thermal storage using shape-stabilized phasechange material plates, *Applied Energy* 84 (2007) 1068–1077.
<http://dx.doi.org/10.1016/j.apenergy.2006.09.005>
- [4] Maha Ahmad , Andre' Bontemps, He'bert Salle'e, Daniel Quenard, Thermal testing and numerical simulation of a prototype cell using light wallboards coupling vacuum isolation panels and phase change material, *Energy and Buildings* 38 (2006) 673–681.
<http://dx.doi.org/10.1016/j.enbuild.2005.07.008>
- [5] Ana Lazaro , Pablo Dolado, Jose M. Marin, Belen Zalba, PCM-air heat exchangers for free-cooling applications in buildings: Empirical model and application to design, *Energy Conversion and Management* 50 (2009) 444–449.

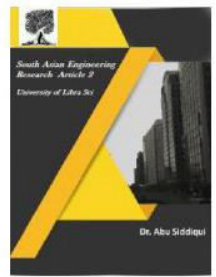


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<http://dx.doi.org/10.1016/j.enconman.2008.11.009>

[6] Katsunori Nagano, Development of the PCM floor supply air-conditioning system, Proceedings of the NATO Advanced Study Institute, Thermal Energy Storage for Sustainable Energy Consumption, 367–373, C_2007 Springer.

[7] Sas̃o Medved, Ciril Arkar, Correlation between the local climate and the free-cooling potential of latent heat storage, Energy and Buildings 40 (2008) 429–437.
<http://dx.doi.org/10.1016/j.enbuild.2007.03.011>

[8] Lv Shilei, Feng Guohui, Zhu Neng, Dongyan Li, Experimental study and evaluation of latent heat storage in phase change materials wallboards, Energy and Buildings 39 (2007) 1088–1091.
<http://dx.doi.org/10.1016/j.enbuild.2006.11.012>

[9] Vincenc Butala, Uros̃ Stritih, Experimental investigation of PCM cold storage, Energy and Buildings 41 (2009) 354–359.
<http://dx.doi.org/10.1016/j.enbuild.2008.10.008>

[10] T. Kondo, S. Iwamoto, Research on Thermal Storage using Rock Wool PCM Ceiling Board, The Tenth International Conference on Thermal Energy Storage, Ecostock 2006 Synopsis, New Jersey

[11] C.K. Halford, R.F. Boehm, Modeling of phase change material peak load shifting, Energy and Buildings 39(2007) 298–305.
<http://dx.doi.org/10.1016/j.enbuild.2006.07.005>

[12] Hed, G., Bellander, R., Mathematical modeling of air heat exchanger. Energy Build. 38 (2006) 82–89. Subramanyam Pavuluri <https://orcid.org/0000-0002-2703-2358>

[13] Arkar , S. Medved, Free cooling of a building using PCM heat storage integrated into the ventilation system, Solar Energy 81(2007) 1078–1087.
<http://dx.doi.org/10.1016/j.solener.2007.01.010>

[14] Pasupathy, L. Athanasius, R. Velraj, R.V. Seeniraj, Experimental investigation and numerical simulation analysis on the thermal performance of a building roof incorporating phase change material (PCM) for thermal management, Applied Thermal Engineering 28 (2008)556–565.
<http://dx.doi.org/10.1016/j.applthermaleng.2007.04.016>

[15] Dariusz Heim, Joe A. Clarke, Numerical modeling and thermal simulation of PCM–gypsum composites with ESP-r, Energy and Buildings 36 (2004) 795–805.
<http://dx.doi.org/10.1016/j.enbuild.2004.01.004>

[16] ASHRAE Handbook Fundamentals 2005