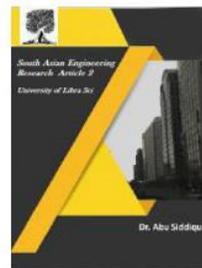




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CFD AND THERMAL ANALYSIS ON AC CONDENSER BY USING DIFFERENT FLUIDS AND DIFFERENT MATERIAL

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ABSTRACT

Air conditioning systems have condenser that removes unwanted heat from the refrigerant and transfers that heat outdoors. The primary component of a condenser is typically the condenser coil, through which the refrigerant flows. Since, the AC condenser coil contains refrigerant that absorbs heat from the surrounding air, the refrigerant temperature must be higher than the air. In this thesis heat transfer by convection in AC by varying the refrigerants are determined by CFD and thermal analysis. The assessment is out on an air-cooled tube condenser of a vapour compression cycle for air conditioning system. The materials considered for tubes are Copper and Aluminium alloys 6061 and 7075. The refrigerants varied will be R 22, R 134 and R407C. CFD analysis is done to determine temperature distribution and heat transfer rates by varying the refrigerants. Heat transfer analysis is done on the condenser to evaluate the better material. 3D modelling is done in CREO and analysis is done in ANSYS.

Keywords: *Condensor, ANSYS, CREO, Thermal analysis*

1. INTRODUCTION

In systems involving heat transfer, a condenser is a device or unit used to condense a substance from its gaseous to its liquid state, by cooling it. In so doing, the latent heat is given up by the substance, and will transfer to the condenser coolant. Condensers are typically heat exchangers which have various designs and come in many sizes ranging from rather small (hand-held) to very large industrial-scale units used in plant processes. For example, a refrigerator uses a condenser to get rid of heat extracted from the interior of the unit to the outside air. Condensers are used in air conditioning, industrial chemical processes such as distillation, steam power plants and other heat-exchange systems. Use of cooling

water or surrounding air as the coolant is common in many condensers.



- A **condenser unit** used in central air conditioning systems typically has a heat exchanger section to cool down and condense incoming refrigerant vapor into liquid, a compressor to raise the pressure of the refrigerant and move it along, and a fan for blowing outside air through the heat

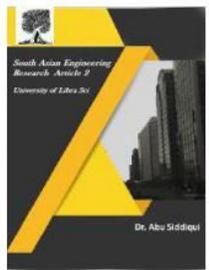


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exchanger section to cool the refrigerant inside. A typical configuration of such a condenser unit is as follows: The heat exchanger section wraps around the sides of the unit with the compressor inside. In this heat exchanger section, the refrigerant goes through multiple tube passes, which are surrounded by heat transfer fins through which cooling air can move from outside to inside the unit. There is a motorized fan inside the condenser unit near the top, which is covered by some grating to keep any objects from accidentally falling inside on the fan. The fan is used to blow the outside cooling air in through the heat exchange section at the sides and out the top through the grating. These condenser units are located on the outside of the building they are trying to cool, with tubing between the unit and building, one for vapor refrigerant entering and another for liquid refrigerant leaving the unit. Of course, an electric power supply is needed for the compressor and fan inside the unit.

- **Direct contact condenser**

In this type of condenser, vapors are poured into the liquid directly. The vapors lose their latent heat of vaporization; hence, vapors transfer their heat into liquid and the liquid becomes hot. In this type of condensation, the vapor and liquid are of same type of substance. In another type of direct

contact condenser, cold water is sprayed into the vapour to be condensed.

Other Types of Condensers

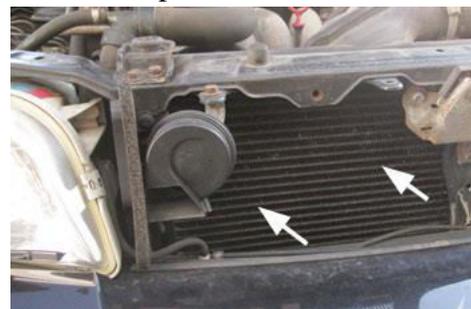
the world of Heating, Ventilation, and Air Conditioning (HVAC), condensers happen to be a topic of great importance. Instead of confusing information, the goal is to provide some basic information on the different types of condensers and their applications.

There are three other condensers used in HVAC systems

- **Water-cooled**
- **Air-cooled**
- **Evaporative**



A/C Condenser problems



The A/C condenser is installed in front of the vehicle, so it often gets damaged in a frontal collision or by rocks or other objects on the road. Corrosion is another enemy of an A/C

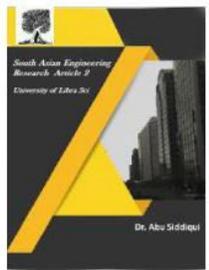


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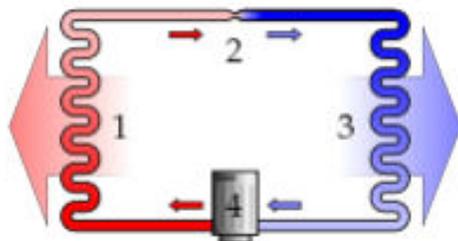


condenser, especially In the places where salt is used on roads in winter months. If a condenser is damaged, the refrigerant leaks out and the air conditioning system stops working. A damaged A/C condenser cannot be repaired and must be replaced. One of the signs of a leaking A/C condenser is an oily greenish residue around the impacted area. Sometimes a leak might be at the place of connection with one of the air conditioning system lines. Automotive repair shops use special equipment to find refrigerant leaks in an air conditioning system. Another issue with the A/C condensers is when the fins get clogged up with leaves and other debris. This reduces the air flow through the condenser fins and affects the efficiency of an air conditioning system.

Air Conditioning System

A simple stylized diagram of the refrigeration cycle:

- 1) Condensing coil,
- 2) Expansion valve,
- 3) Evaporator coil,
- 4) Compressor.



Refrigeration cycle

In the refrigeration cycle, a heat pump transfers heat from a lower-temperature heat source into a higher-temperature heat sink. Heat would naturally flow in the opposite direction. This is the most common type of air conditioning. A refrigerator works in

much the same way, as it pumps the heat out of the interior and into the room in which it stands. This cycle takes advantage of the way phase changes work, where latent heat is released at a constant temperature during a liquid/gas phase change, and where varying the pressure of a pure substance also varies its condensation/boiling point.

By placing the condenser (where the heat is rejected) inside a compartment, and the evaporator (which absorbs heat) in the ambient environment (such as outside), or merely running a normal air conditioners refrigerant in the opposite direction, the overall effect is the opposite, and the compartment is heated. This is usually called a heat pump, and is capable of heating a home to comfortable temperatures (25 °C; 70 °F), even when the outside air is below the freezing point of water (0 °C; 32 °F).

2.COOLING LOAD CALCULATIONS

Width = 2.94m
 Length = 9.06m
 Height = 2.47m
 Window = 1.36 × 1
 Wall thickness = 0.25m
 Door sizes = 0.8 × 1
 East wall = 9.06 × 2.47 = 22.3'
 West wall = 9.06 × 2.47 = 22.3'
 South wall = 2.94 × 2.47 = 7.2'
 North wall = 2.94 × 2.47 = 7.2'
 West door = 0.8 × 1.9 = 1.52
 North window = 1.36 × 1.18 = 1.6048
 Bulbs = 6 × 40w = 240w
 Floor volume = length × width × hei
 = 9.06 × 2.94 × 2 = 65.791m³
 Door area = w × h = 0.8 × 1.98 = 1.584
 Wall thickness = 0.254m
 No of systems = 20
 Window area = 1.36 × 1.18 = 1.6048
 No of windows = 1 = 1.6048m²
 No of lights = 6 = 6 40 = 240w
 Florescent coefficient = 1.25
 Total lighting load = 240 × 1.25 = 300w
 Solar heat gain factor = (SHGF)



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South wall = 140 w/m^2
 North wall = 120 w/m^2
 West wall = 340 w/m^2
 East wall = 60 w/m^2
 Overall coefficient of heat transfer (U) $\text{w/m}^2\text{K}$
 $U_{\text{wall}} = 1.56 \text{ w/m}^2\text{K}$
 $U_{\text{roof}} = 5.675 \text{ w/K}$
 $U_{\text{floor}} = 159 \text{ w/K}$
 $U_{\text{door}} = 142 \text{ w/K}$
 $U_{\text{window}} = 4.70 \text{ w/m}^2\text{K}$
 Equivalent temperature differences ()
 $t_e \text{ of north wall} = 90^\circ$
 $t_e \text{ of south wall} = 11^\circ$
 $t_e \text{ of west wall} = 11^\circ$
 $t_e \text{ of east wall} = 6^\circ$
 $t_e \text{ of roof} = 19^\circ$
 $t_e \text{ of floor} = 2.4^\circ$
 No of persons = 40
 Sensible heat load per person = 117W
 Latent heat load per person = 50w
 Ventilation required per person = $0.28 \text{ m}^3/\text{min}$
 Out door conditions:
 Dry bulb temperatures = 38°C RH 60%
 $W: 0.011 \text{ kg/kg}$ of dry air ratio

3. 3D MODELING OF CONDENSER

THE MODEL IS DESIGNED FROM
 BASED ON JOURNAL OF PLATE-FIN-
 AND-TUBE CONDENSER
 PERFORMANCE AND DESIGN FOR
 REFRIGERANT R-410A AIR-
 CONDITIONER

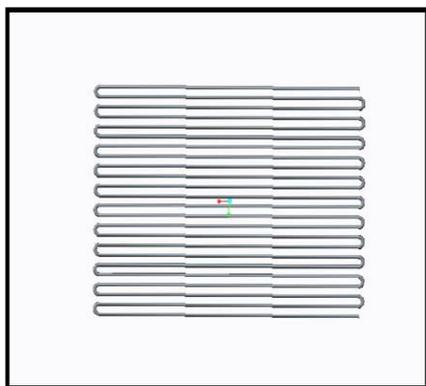


Fig - 3D Model

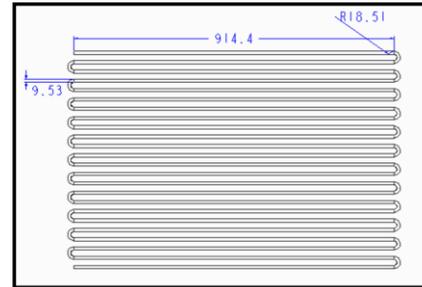


Fig – 2D Drafting

4. CFD ANALYSIS FOR CONDENSER

Save Creo Model as .iges format

→Ansys → Workbench→ Select analysis system → Fluid Flow (Fluent) → double click

→→Select geometry → right click → import geometry → select browse →open part → ok



Fig - Imported model from Creo 2.0

→→ Select mesh on work bench → right click →edit

Select mesh on left side part tree → right click → generate mesh →

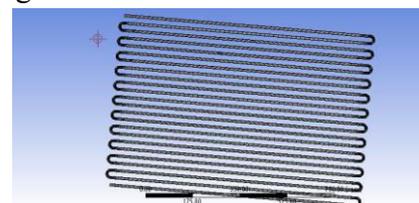


Fig – Meshed Model



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SPECIFYING BOUNDARIES FOR INLET AND OUTLET

Inlet

Select edge → right click → create named section → enter name → inlet

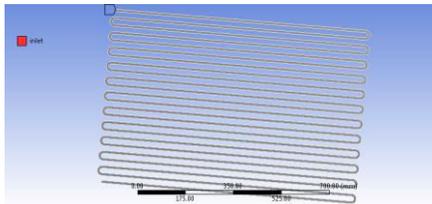


Fig – Fluid inlet

Outlet

Select edge → right click → create named section → enter name → outlet

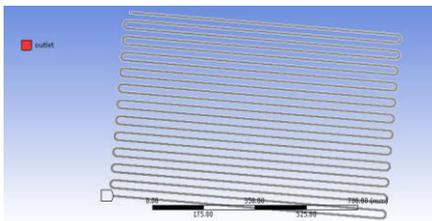
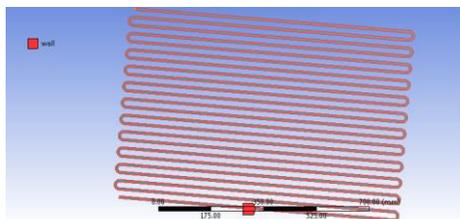
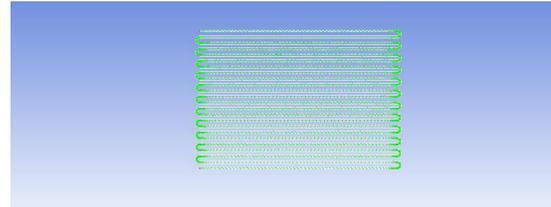


Fig – Fluid outlet

Wall



File export → fluent → input file (mesh) → save required name → save.
→→ Ansys → fluid dynamics → fluent → select working directory → ok
→→ file → read → mesh → select file → ok.



General → Pressure based
Model → energy equation → on.
Viscous → edit → k-epsilon

FLUID - R22

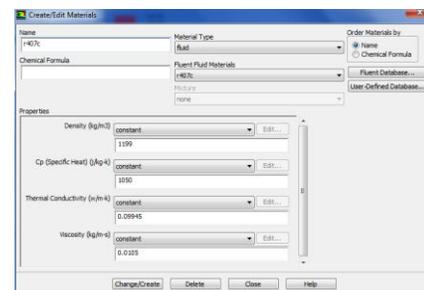
| Temperature - T (°C) | Density - ρ (kg/m³) | Specific Heat Capacity - Cp (10³ J/kg.K) | Thermal Conductivity - k (W/m.K) | Kinematic Viscosity - ν (10⁻⁶ m²/s) | Prandtl Number - Pr |
|----------------------|---------------------|--|----------------------------------|-------------------------------------|---------------------|
| -50 | 1547 | 0.875 | 0.067 | 0.310 | 6.2 |
| -40 | 1519 | 0.885 | 0.069 | 0.279 | 5.4 |
| -30 | 1490 | 0.896 | 0.069 | 0.253 | 4.8 |
| -20 | 1461 | 0.907 | 0.071 | 0.235 | 4.4 |
| -10 | 1429 | 0.920 | 0.073 | 0.221 | 4.0 |
| 0 | 1397 | 0.935 | 0.073 | 0.214 | 3.8 |
| 10 | 1364 | 0.950 | 0.073 | 0.203 | 3.6 |
| 20 | 1330 | 0.966 | 0.073 | 0.198 | 3.5 |
| 30 | 1295 | 0.984 | 0.071 | 0.194 | 3.5 |
| 40 | 1257 | 1.002 | 0.069 | 0.191 | 3.5 |
| 50 | 1216 | 1.022 | 0.069 | 0.190 | 3.5 |

FLUID - R134A

R-134a Properties - SI Units

| Temperature (°C) | Absolute Pressure (bar) | Density Liquid (kg/m³) | Density Vapor (kg/m³) | Enthalpy Liquid (kJ/kg) | Enthalpy Vapor (kJ/kg) | Entropy Liquid (kJ/kgK) | Entropy Vapor (kJ/kgK) |
|------------------|-------------------------|------------------------|-----------------------|-------------------------|------------------------|-------------------------|------------------------|
| -50 | 0.15935 | 1472.0 | 0.5291 | 24.491 | 261.491 | 0.68772 | 1.8014 |
| -50 | 0.29477 | 1444.9 | 1.6526 | 36.302 | 267.779 | 0.74358 | 1.7809 |
| -40 | 0.51225 | 1419.0 | 2.773 | 48.631 | 274.068 | 0.79756 | 1.7648 |
| -30 | 0.84379 | 1388.2 | 4.4307 | 61.130 | 280.324 | 0.84995 | 1.7514 |
| -20 | 1.32119 | 1358.4 | 6.7903 | 73.833 | 286.513 | 0.901 | 1.7413 |
| -10 | 2.00575 | 1327.4 | 10.047 | 86.777 | 292.598 | 0.95095 | 1.7309 |
| 0 | 2.92769 | 1295.1 | 14.433 | 100.0 | 298.536 | 1 | 1.7204 |
| 10 | 4.14571 | 1261.2 | 20.226 | 113.540 | 304.276 | 1.04834 | 1.72196 |
| 20 | 5.71665 | 1225.5 | 27.773 | 127.437 | 309.756 | 1.09613 | 1.71806 |
| 30 | 7.70132 | 1187.5 | 37.517 | 141.736 | 314.892 | 1.14354 | 1.71473 |
| 40 | 10.1848 | 1148.7 | 50.055 | 156.491 | 319.575 | 1.19073 | 1.71152 |
| 50 | 13.1773 | 1109.2 | 66.204 | 171.778 | 323.652 | 1.23704 | 1.70792 |
| 60 | 16.8156 | 1062.9 | 87.346 | 187.715 | 326.896 | 1.28248 | 1.70325 |
| 70 | 21.1668 | 996.49 | 115.664 | 204.515 | 328.941 | 1.3339 | 1.69665 |
| 80 | 26.3396 | 928.78 | 155.130 | 222.616 | 329.095 | 1.38434 | 1.68885 |
| 90 | 32.4499 | 838.51 | 216.906 | 243.168 | 328.655 | 1.42978 | 1.68032 |
| 100 | 39.7254 | 649.71 | 367.064 | 273.641 | 309.037 | 1.5198 | 1.61486 |

FLUID - R407C



5. THERMAL ANALYSIS MATERIAL – ALUMINIUM 6061 FLUID –R22

Open work bench 14.5>select **steady state thermal** in analysis systems>select geometry>right click on the

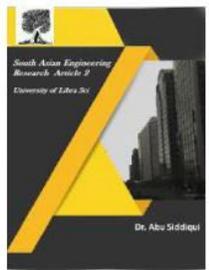


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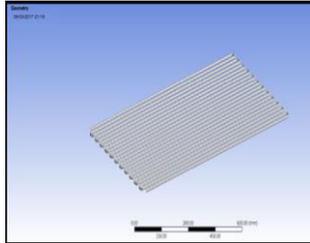


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geometry>import geometry>select IGES
file>open

Imported model



ALUMINIUM 6061 MATERIAL PROPERTIES

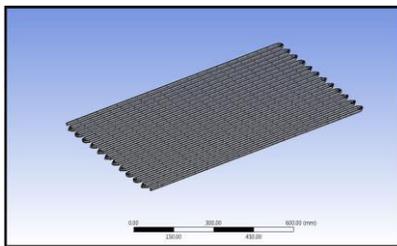
Thermal conductivity of aluminum = 15.1W/mk

Specific heat =356J/Kg K

Density = 0.00000412 Kg/mm³

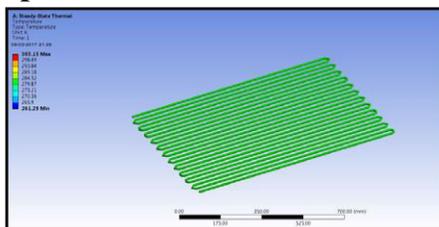
Model >right click>edit>select generate mesh

Meshed model

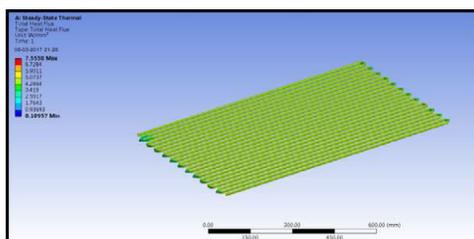


Results

Temperature

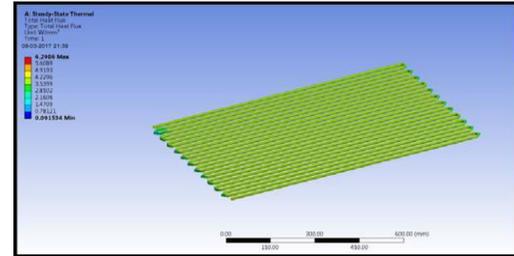


Heat flux



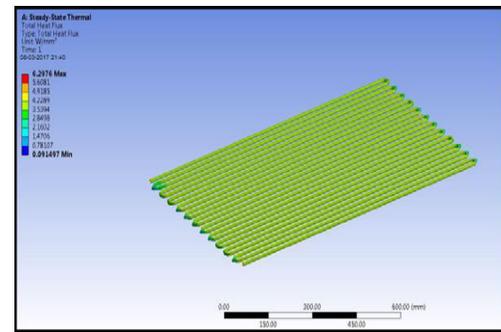
FLUID - R134A

Heat flux



FLUID - R407C

Heat flux



MATERIAL -COPPER

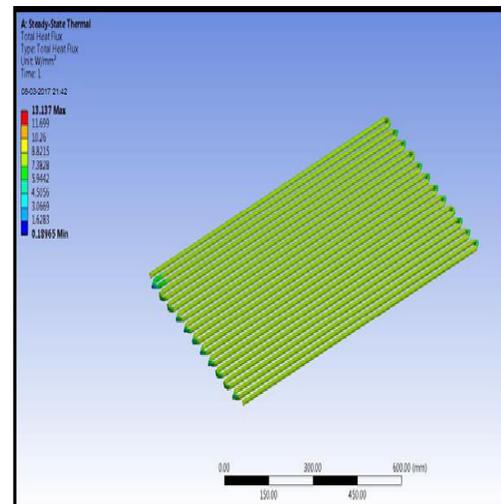
FLUID -R22

Thermal conductivity of aluminum = 59.1W/mK

Specific heat =421 J/Kg K

Density = 0.00000771Kg/mm³

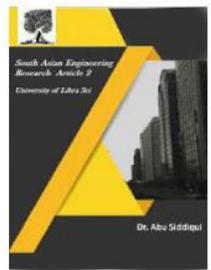
Heat flux



FLUID - R407C

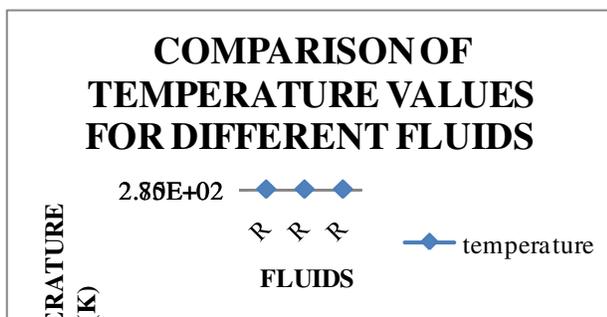
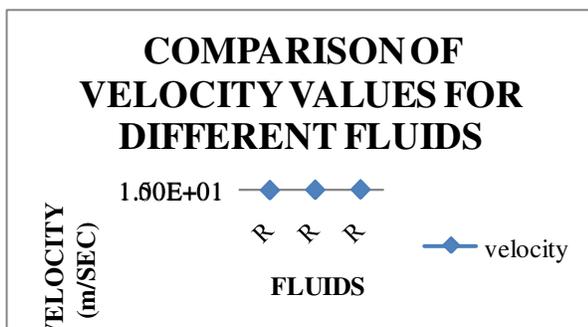
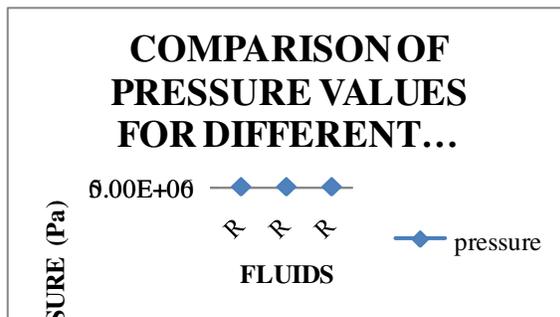
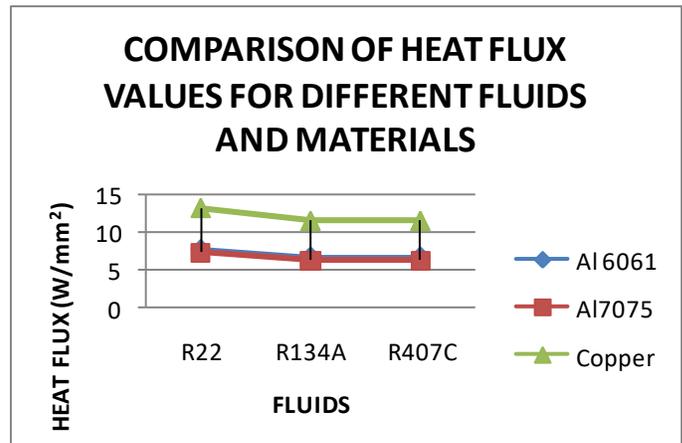


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6. RESULTS AND DISCUSSION CFD ANALYSIS

| Fluids | Pressure (Pa) | Temperature (K) | Velocity (m/Sec) | Wall Function Heat Transfer Coefficient (W/m ² -K) | Mass Flow Rate (Kg/Sec) | Total Heat Transfer Rate (W) |
|--------|---------------|-----------------|------------------|---|-------------------------|------------------------------|
| R22 | 3.12e+06 | 2.80e+02 | 1.26e+01 | 1.08e+02 | 0.0278429 | 855.549 |
| R134A | 3.37e+06 | 2.83e+02 | 1.36e+01 | 1.56e+02 | 0.013374329 | 181.78711 |
| R407C | 1.52e+06 | 2.83e+02 | 1.47e+01 | 1.47e+02 | 0.01423645 | 48.704102 |



7. CONCLUSION

In this thesis heat transfer by convection in AC by varying the refrigerants are determined by CFD and thermal analysis. The assessment is out on an air-cooled tube condenser of a vapour compression cycle for air conditioning system.

The materials considered for tubes are Copper and Aluminum alloys 6061 and 7075. The refrigerants varied will be R 22, R 134 and R407C. CFD analysis is done to determine temperature distribution and heat transfer rates by varying the refrigerants. Heat transfer analysis is done on the condenser to evaluate the better material.

By observing CFD analysis results, the heat transfer coefficient is more when R134A is used and heat transfer rate is more when R22 is used than other fluids. By observing thermal analysis results, the heat flux is more when R22 is used and when material Copper is used. (i.e) The heat transfer rate is more when fluid R134A and material Copper is used.

8. REFERENCES

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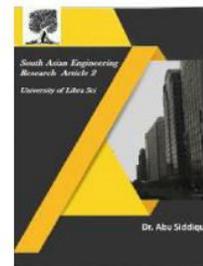


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- Based Cooling System by Balaji N, Suresh Mohan kumar P
- [2]. EFFICIENT USAGE OF WASTE HEAT FROM AIR CONDITIONER by M. Joseph Stalin, S. Mathana Krishnan, G. Vinoth Kumar
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